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Rolling Ring Drives: Catalog RG/RGK/KI



Non Contact Flange Detecting System: Catalog FA



Guide System: Catalog GS



Linear Drive Nut: Catalog RS



Timing Belt Drive: Catalog AZ



Fast Action Clamping System Uhing-easylock®: Catalog UE



Smooth Shaft Fastener U-Clip: Catalog UE



Engineering: Catalog EG



Automatic Winding Width Control: Catalog AVS



Joachim Uhing KG

GmbH & Co. - the originator of the Rolling Ring Principle - successful for over 50 years.

More about us at: www.uhing.com

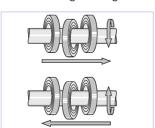
Our worldwide network of agencies guarantees a reliable service on the spot.

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Uhing - Linear Drive Nut

The Actuator for:

- Linear Motion Drives
- Measurement and Inspection Technology
- Materials Handling Systems
- Control Systems
- Medical Engineering





Uhing-Linear Drive Nuts (RS) are non-positive drives which convert the rotation of a plain round shaft into linear motion.

Principle of operation

The effect is achieved by pressing specially crowned Rolling Rings against the shaft and allowing them to roll on the surface of the shaft at an angle which determins their pitch.

DIST. AUTORIZADO QRO (442) 1 95 72 60 ventas@industrialmagza.com



With permission of Zeiss



With permission of Zoller

Endswitches for

speed control and clutch

Repeating



With permission of DMG Microset GmbH



- Backlash-free
- Resistant to vibrations
- Compact design
- Overload protection
- High-efficiency
- Quiet in operation
- Low maintenance
- Free-movement lever
- Good sealing possibilities
- Linked nuts for higher side thrust
- Left- and right-hand pitch on the same shaft

Adjustment Uhing Linear Thrust setting rod Drive Nut Screw of Linear Drive Nut

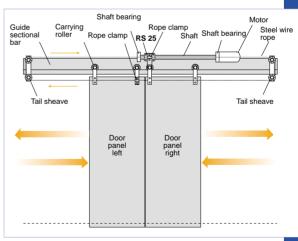


freemovement

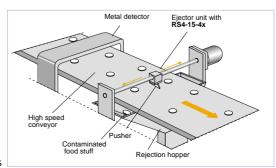


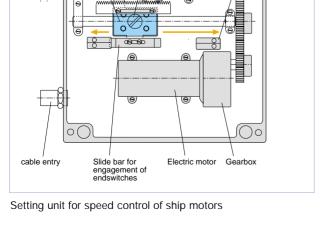
linked nuts

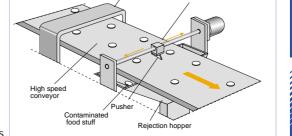
left- and right hand pitch



Drive for double sliding doors







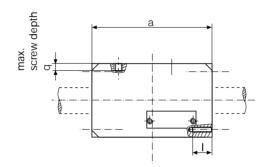
Dimensions and Technical Details

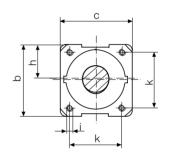
Uhing - Linear Drive Nurpustrial TIAGZA* MEX (55) 53 63 23 31 MTY (81) 83 54 10 18 Output DIST. AUTORIZADO QRO (442) 1 95 72 60 ventas@industrialmagza.com **Types RS**

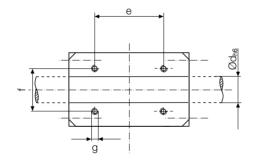




Special design for open Linear Drive Nuts upon request







The CAD - drawings and technical details are available in the Internet.

Туре	Weight m (kg)	a*	Dime a ₁ *	nsions b	for RS	Type d _{h6}	n (mm e) f	g	h±0.3	i	k	I	q	Technic F _{RS} (N)	al Details M₀(Ncm)	h(mm)
RS3-10-4	0,14	47	65	35	35	10	30	18	M 4	17,3	M 3	26	6	5	100	1,8	5,0
RS4-10-4	0,18	55	73	"	и	п	"	"	n	11	"	"	"	п	200	5,0	"
RS4-15-4	0,23	62	82	40	40	15	26	18	M 4	19,6	M 4	30	8	5	260	5,0	7,5
RS4-20-4	0,55	83	108	52	52	20	40	30	M 5	26	M 5	40	11	8	420	10,0	10,0
RS4-25-4	0,70	85	110	60	60	25	40	30	M 5	29,4	M 5	45	10	9	600	20,0	12,5
RS4-35-4	1,55	105	126	80	80	35	50	40	M 6	40	M 6	60	12	13	900	45,0	17,5

M 8

M10

48.8

58,4

Heavy print: standard versions

RS4-50-3

RS4-60-3

Dimensions

2,70

4,20

*Attention:

100 100 50

120 120 60

120 140

130 156

If wipers are used, dimension a becomes a₁.

50 50

69 62

15 F_{RS} (N) M_o (Ncm)

16

= Maximum available side thrust = Idling torque = Maximum pitch

1300

2000

140,0

200,0

25,0

30,0





Features

F Free-Movement Lever, mechanical

when operated, the nut can be slide freely along the shaft



P Free-Movement Lever, pneumatical

as above, operated with a pressure of

p = 6 bar Attention: Units supplied with **P** have a reduced thrust. Refer to supplier for details



Adapter

for twist free coupling system



Wipers

for sealing between nut and shaft (to +70°C)

Attention! For units with wipers please note dimension a₁ on page 4.



R Steady Rollers

Rolls on Linear Drive Nut (in conjunction with a guide bar) prevent the rotation of the nut on the shaft.



Details of other types upon request.

Product Survey

(Type designation chart)

Product group	Uhing Linear Drive Nut					ıt	
Type reference	RS						
Style (Number of Rolling Rings)	3 o. 4 3 or 4						
Size (Shaft diameter)	10	15	20	25	35	50	60
Design category	4	4	4	4	4	3	3
Pitch direction L (= left), R (= right)							
Pitch max.	0,1 · 0),2 - (0,3 · (0,4 · (),5 x	shat	ft Ø
Available features ¹⁾	F, P, R						
Customer specific features ²⁾	Х						

Heavy print: standard versions

1) Available features

Free Movement Lever

F mechanical

P pneumatical

R Steady Rollers

²) Customer specific features

Adapter

• Wipers • Felt rings

- Increased protection against corrosion
- Specific pitch
- Grease nipples Reduced thrust

Ordering specifications / Example

Type reference	RS	4 -	- 25 -	- 4	R	12.5	Р	Х
Style								
Size (shaft dia.) Separator symbols								
Design category								
Pitch direction								
Pitch								
Available features								
Customer specific features								



If you wish Joachim Uhing KG GmbH & Co. to many place of your application, place of your application of your application, place of your application of your applicatio

Formulae and related units

 $\begin{array}{ll} d(mm) & = shaft \ diameter \\ F(N) & = side \ thrust \ required \\ F_{RS}(N) & = side \ thrust \ produced \ by \\ & Linear \ Drive \ Nut \ type \ RS \end{array}$

 $F_R(N) = \begin{array}{ll} \text{force of friction } (F_N \cdot \mu) \\ \text{only relevant when the} \\ \text{associated mass is mounted} \\ \text{on its own independent} \\ \text{carriage} \end{array}$

F_N(N) = normal force of total weight of associated mass and carriage

 $\begin{array}{ll} \mu & = \text{coefficient of friction} \\ F_Z(N) & = \text{additional force e. g.} \\ & \text{component of the cutting} \\ & \text{force of a separator} \end{array}$

 $\begin{array}{ll} f(mm) &= \text{ shaft sag from diagram} \\ g(m/s^2) &= \text{ acceleration due to gravity} \\ &\quad (9.81 \text{ m/s}^2). \text{ Note: for} \\ &\quad \text{ horizontal applications} \\ &\quad m \cdot g = 0 \end{array}$

h(mm) = Drive Nut pitch (travel per shaft revolution)

I(mm) = length of shaft between centre of bearing bracketsm(kg) = total mass to be moved,

 total mass to be moved, including Drive Nut, connections etc. Md (Ncm) = drive torque Mo (Ncm) = idling torque

n(r.p.m.) = shaft speed $n_{crit}(r.p.m.)$ = critical shaft speed

P(kW) = drive power required t(s) = acceleration or braking time

v(m/sec) = maximum speed of travel

C(N) = dynamic loading of Rolling Rings P-(N) = radial loading of

P_R(N) = radial loading of Rolling Rings

1. Side thrust

$$F = 2\left(\frac{m \cdot v}{t} + m \cdot g\right) + F_R + F_Z$$

A Drive Nut should be selected which has a greater side thrust than the value calculated.

F< F_{RS}

Several smaller Drive Nuts can be coupled together if available space so dictates. The total thrust available is the sum of the individual values.

2. Shaft speed

$$n = \frac{v \cdot 6 \cdot 10^4}{h_{max}}$$

2.1. Max. shaft speed

RS 3-10-4 =10000 min⁻¹

RS $4-15-4 = 8000 \text{ min}^{-1}$

RS $4-20-4 = 7000 \text{ min}^{-1}$

RS $4-25-4 = 6000 \text{ min}^{-1}$

RS $4-35-4 = 4000 \text{ min}^{-1}$

RS $4-50-3 = 3400 \text{ min}^{-1}$

RS $4-60-3 = 2500 \text{ min}^{-1}$

2.2 Critical shaft speed

$$n_{crit} = 1,225 \cdot 10^8 \frac{d}{l^2}$$

Note

Depending upon its quality, the shaft can go out of balance at a speed of up to 25 % lower than that specified above.

If it is necessary to go through a critical range in order to reach the operational speed, this can lead to short term shaft vibration. This has no effect on the operation of the Drive Nut.

If the operational speed is in the critical speed range, this can be rectified as follows:

- 1. with a double bearing support at one end, increase factor approx. 1,5.
- 2. with double bearing supports at both ends, increase factor approx. 2,2.

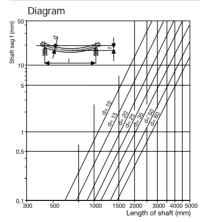
The distance between the bearing support brackets should be at least 2.5 x the diameter of the shaft.

3. Drive torque

$$M_d = \frac{F_{RS} \cdot h}{20 \cdot \pi} + Mo$$

Values for Mo to be taken from the technical detail tables.

4. Shaft sag



5. Calculation of the operational life of Rolling Rings

. Select C	Тур	C(N)
	RS 10	4620
	RS 15	5590
	RS 20	9360
	RS 25	11200
	RS 35	15900
	RS 50	21600
	DS 60	20600

RS 10 :
$$P_R = 5 \cdot F_{RS}^*$$

RS 15 - 60: $P_R = 2.5 \cdot F_{RS}^*$

*F = <u>calculated</u> value of the side thrust according to 1. only if increasing of operational life time of the rolling rings is really necessary. In case of order it is an absolute must to mention.

3. Divide C by P_R

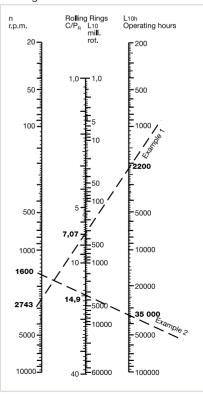
4. Calculate the required shaft speed

$$n = \frac{v \cdot 6 \cdot 10^4}{h_{max}}$$

5. Determine the operational life in hours from the nomogram

Example 1	Example 2
RS4-35-4R17,5 speed 0,8 m/s	RS4-15-4R7,5 reduced side thrust 150 N speed 0,2 m/s
1. C = 15900	C = 5590
2. P _R = 2,5 · 900 N = 2250 N	$P_R = 2.5 \cdot 150 \text{ N} = 375 \text{ N}$
$3. \frac{C}{P_R} = \frac{15900}{2250} = 7.07$	$\frac{C}{P_R} = \frac{5590}{375} = 14,9$
4. n = $\frac{0.8 \cdot 6 \cdot 10^4}{17.5}$ = 2743 r.p.m.	$n = \frac{0.2 \cdot 6 \cdot 10^4}{7.5} = 1600 \text{ r.p.m.}$
5. L _{10h} = 2200 operating hours	L_{10h} = 35000 operating hours

Nomogram





1. Shaft material

1.1. Basic requirements

Uhing Linear Drives should only be used in conjunction with steel shafts manufactured from induction surface hardened, ground and finished bar of the following quality, minimum:

- surface hardness: 50 HRC
- tolerance on diameter: h6
- out of roundness: maximum one half of the diameter variation permitted by ISO tolerance h6
- true running tolerance (DIN ISO 1101): ≤ 0,1 mm/m

1.2. Uhing - precision shaft

Standard:

Material Cf 53, Mat.-Nr. 1.1213, induction surface hardened, 60-64 HRC

Rust resistant:

Material X 40 Cr 13, Mat.-Nr. 1.4034, induction surface hardened, 51-55 HRC

Rust and acid resistant: Material X 90 CrMoV 18, Mat.-Nr. 1.4112, induction surface hardened. 52-56 HRC

- all ground and superfinished
- surface roughness: mean value (DIN 4768 T.1) R_a:≤ 0.35 µm
- tolerance on diameter: h6
- out of roundness: maximum one half of the diameter variation permitted by ISO tolerance h6
- true running tolerance (DIN ISO 1101): ≤ 0.1 mm/m

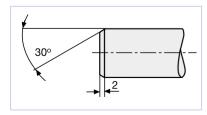
1.3. Uhing precision shafts with enhanced true running tolerance

Available in the above styles, but

true running tolerance (DIN ISO 1101): ≤0.03 mm/m

1.4. Leading end chamfer

The leading end of the shaft should be chamfered to avoid damage to the Rolling Rings when screwing the unit onto the shaft.



2. Pitch

The standard pitch is 0,5 x d. This can be ordered for RS as either a right- or a left-handed pitch.

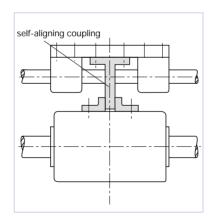
Unless otherwise specified, units having a right-handed pitch will be supplied. For RS subsequent alterations to the pitch are possible with units having a design category -4 reference by changing the associated pitch control wedges.

Non-standard pitches 0,1 - 0,2 - 0,3 and 0,4 x d are available. In this version reduction of the side thrust is recommended to improve the smooth running.

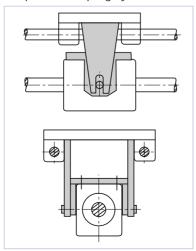
3. Separately carried loads

If Uhing Linear Drive Nuts are used to move separately carried loads, allowance should be made in the coupling to compensate for any misalignment between the drive

shaft and the carriage. The available side thrust will otherwise be affected. If the application so permits, we recommend the use of our twist-free coupling system.



Torque-free coupling system



4. Vertical applications

For vertical applications we advise the use of a directly braked motor so as to avoid the possibility of the shaft rotating backwards and the Drive Nut falling due to the high efficiency of the drive.

Depending upon the application (safety considerations and value of the installation) a reserve of side thrust should be built in (using a second Drive Nut).

With units having a free-movement lever, care must be taken before its operation to ensure that they are unable to drop in an uncontrolled manner - danger of injury!

5. Temperature range

Uhing Linear Drive Nuts are suitable for operation at temperatures from -10°C to +70°C

Please enquire for other temperatures.

6. Maintenance

For the lubrication of the shaft, commercially available MoS2-free ballbearing greases can be used, e. g. SKF Alfalub LGMT2, Shell Alvania R2 or G2, Esso Beacon 2, BP Energrease

Procedure:

Clean the shaft and spread the grease as thinly as possible with a rag. Frequency:

Once every ten weeks.

7. Symmetry

The maximum difference in pitch for the two directions of travel is 2 %. We therefore recommend the use of positional sensors for positioning applications.

We reserve the right to make technical alterations.

For further information please refer to our Operating Instructions 05e, available on request or in the internet as download:

www.uhing.com





Uhing

worldwide service





The adresses of our agencies are available in the Internet:

www.uhing.com

Joachim Uhing KG GmbH & Co.

Kieler Straße 23 D-24247 Mielkendorf Phone +49 04347 - 906-0 +49 04347 - 906-40 e-mail: sales@uhing.com www.uhing.com Germany

